

# SIMPLE HARMONIC MOTION

## Conditions:

- Force/acceleration is proportional to and in the opposite direction to displacement.

$$a = -\omega^2 x$$

\* Amplitude: max displacement

= from equilibrium:  $x_{max} = \omega^2 A$

\* Displacement at any point:

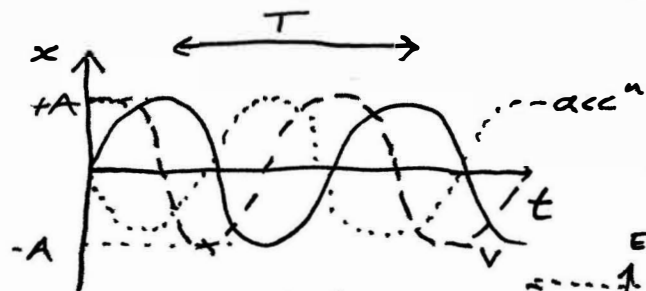
$$x = A \sin(2\pi ft) \quad (\text{calc: RAD!})$$

(cos)

\* Velocity at any point:

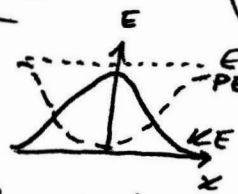
$$v = \pm 2\pi f \sqrt{A^2 - x^2}$$

$$v_{max} = 2\pi f A = \omega A \quad (x=0) \text{ at eqm}$$



\* To get  $v_{max}$ ,  $2\pi f A = \frac{2\pi A}{T}$   
or gradient at eqm.

\* Total E = max KE =  $\frac{1}{2} m v_{max}^2$   
or max PE



Simple Pendulum:

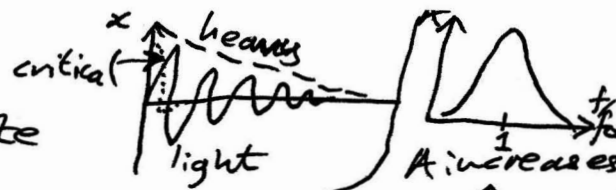
$$T = 2\pi \sqrt{\frac{L}{g}}$$

← to centre of bob

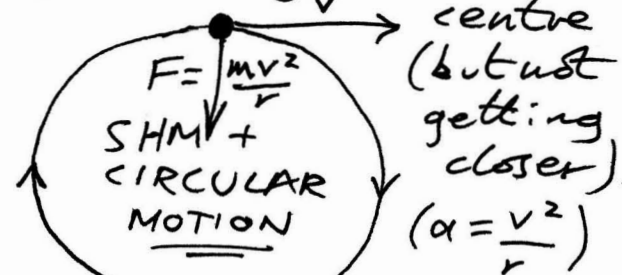
\* If given height of pendulum  
GPE = KE

Damping  
When external force opposes motion (opposite dir to v)

Resonance: when f of external driving matches natural f.  
 $\frac{\pi}{2}$  rad / 90° out of phase with



Force always perpendicular to velocity. Constant speed, but always accelerating towards



$\omega$ : angular vel / freq / speed  
radians per second ( $\text{rads}^{-1}$ )

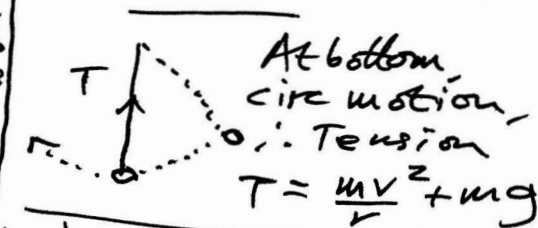
$$v = \frac{2\pi r}{T} = (2\pi f)r \quad \therefore v = \omega r$$

$$\omega \quad \therefore F = m\omega^2 r$$

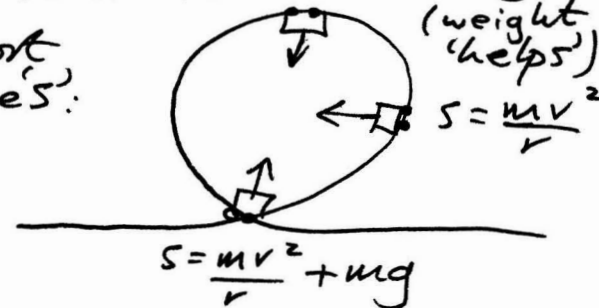
Spring:

$$T = 2\pi \sqrt{\frac{m}{k}}$$

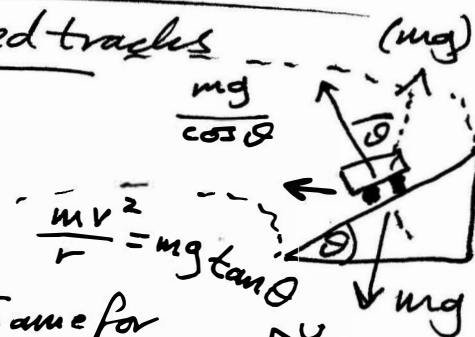
Pendulum tension:



If vertical:  $S = \frac{mv^2}{r} - mg$   
support force S: (weight 'helps')



Banked tracks



Same for banked plane

$$\frac{mv^2}{r} = mg \tan \theta$$

x2 for springs in parallel  
÷2 " " " series

Video